



(51) International Patent Classification:
F01K 23/10 (2006.01) *F01K 23/16* (2006.01)

(21) International Application Number:
PCT/NL2017/050505

(22) International Filing Date:
26 July 2017 (26.07.2017)

(25) Filing Language: Dutch

(26) Publication Language: English

(30) Priority Data:
2017247 28 July 2016 (28.07.2016) NL

(72) Inventor; and
(71) Applicant: **OUWERKERK, Henk** [NL/NL]; Saxofoon-
straat 8, 5702 KC Helmond (NL).

(74) Agent: **ALGEMEEN OCTROOI - EN MERKENBU-
REAU B.V.**; P.O. Box 645, 5600 AP Eindhoven (NL).

(81) Designated States (unless otherwise indicated, for every
kind of national protection available): AE, AG, AL, AM,
AO, AT, AU, AZ, BA, BB, BG, BH, BN, BR, BW, BY, BZ,
CA, CH, CL, CN, CO, CR, CU, CZ, DE, DJ, DK, DM, DO,
DZ, EC, EE, EG, ES, FI, GB, GD, GE, GH, GM, GT, HN,
HR, HU, ID, IL, IN, IR, IS, JO, JP, KE, KG, KH, KN, KP,
KR, KW, KZ, LA, LC, LK, LR, LS, LU, LY, MA, MD, ME,
MG, MK, MN, MW, MX, MY, MZ, NA, NG, NI, NO, NZ,

OM, PA, PE, PG, PH, PL, PT, QA, RO, RS, RU, RW, SA,
SC, SD, SE, SG, SK, SL, SM, ST, SV, SY, TH, TJ, TM, TN,
TR, TT, TZ, UA, UG, US, UZ, VC, VN, ZA, ZM, ZW.

(84) Designated States (unless otherwise indicated, for every
kind of regional protection available): ARIPO (BW, GH,
GM, KE, LR, LS, MW, MZ, NA, RW, SD, SL, ST, SZ, TZ,
UG, ZM, ZW), Eurasian (AM, AZ, BY, KG, KZ, RU, TJ,
TM), European (AL, AT, BE, BG, CH, CY, CZ, DE, DK,
EE, ES, FI, FR, GB, GR, HR, HU, IE, IS, IT, LT, LU, LV,
MC, MK, MT, NL, NO, PL, PT, RO, RS, SE, SI, SK, SM,
TR), OAPI (BF, BJ, CF, CG, CI, CM, GA, GN, GQ, GW,
KM, ML, MR, NE, SN, TD, TG).

Published:
— with international search report (Art. 21(3))
— before the expiration of the time limit for amending the
claims and to be republished in the event of receipt of
amendments (Rule 48.2(h))

(54) Title: STEAM AND GAS TURBINE DEVICE

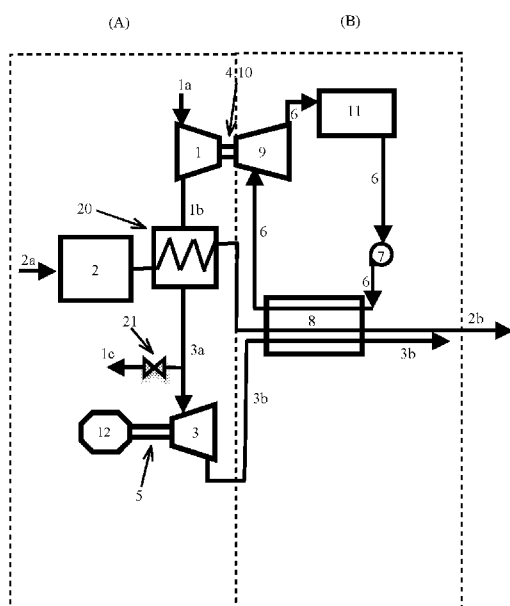


Fig. 1a

(57) Abstract: The invention relates to a steam and gas turbine device comprising a gas turbine unit (A) and a steam turbine unit (B), wherein the gas turbine unit (A) is partly composed of at least the following: a gas compressor (1) positioned on a compressor axle (4), a combustion chamber (2) and a gas turbine (3) positioned on a gas turbine axle (5); and wherein the steam turbine unit (B) is partly composed of the following: a closed steam line (6) in which at least the following are accommodated: a pump (7); a steam converter (8; 50), which, in operation, is in heat-exchanging contact with the gases (3b) coming from the gas turbine (3); a steam turbine (9) positioned on a steam turbine axle (10); as well as a condenser (11).



Brief title: Steam and gas turbine device.

DESCRIPTION

The invention relates to a steam and gas turbine device comprising a gas turbine unit (A) and a steam turbine unit (B), wherein the gas turbine unit (A) is partly composed of at least the following: a gas compressor positioned on a compressor axle, a combustion chamber and a gas turbine positioned on a gas turbine axle; and wherein the steam turbine unit (B) is partly composed of the following: a closed steam line in which at least the following are accommodated: a pump; a steam converter, which, in operation, is in heat-exchanging contact with the gases coming from the gas turbine; a steam turbine positioned on a steam turbine axle; as well as a condenser.

Steam and gas turbine devices are well known and are used industrially in all kinds of different configurations. An example of a steam and gas turbine device according to the abovementioned introduction is disclosed, for example, in Netherlands patent no. 1028373C1.

The abovementioned known variants of known steam and gas turbine devices are relatively complicated devices with a number of constructional drawbacks and limitations with regard to transferring the power generated to a load, as a result of which the overall efficiency of such installations is also limited.

A steam and gas turbine device according to the introduction of Claim 1 is disclosed, for example, in German patent publication no. DE102010033659, or British patent publication no. GB671702.

The object of the present invention is to provide a steam and gas turbine device according to the abovementioned introduction which does not have the drawbacks associated with the multi-axle and single-axis variant. More particularly, it is an object of the invention to provide a steam and gas turbine device of simple construction and without all kinds of complicated additional components.

According to the invention, the steam and gas turbine device according to the invention is characterized by the fact that the gas turbine unit (A) furthermore comprises at least one heat exchanger, which brings the combustible flue gases coming from the combustion chamber into heat-exchanging contact with the gases coming from the gas compressor.

As a result thereof, it is possible to deploy the device in a more functional manner and also to implement it in operating conditions with only a combustion chamber.

More particularly, the steam converter comprises a steam generator,

which, in operation, is in heat-exchanging contact with the combustible flue gases coming from the combustion chamber (2).

With a further functional embodiment, the device furthermore comprises a first air pre-heater, which brings the steam line into heat-exchanging contact with the gases coming from the gas compressor. As a result thereof, the compressed gases coming from the gas compressor are pre-heated by the steam in the steam line, which significantly improves the efficiency of the heat exchanger and the gas turbine.

In this case, the first air pre-heater may be accommodated in the device downstream viewed from the steam generator and upstream viewed from the at least one heat exchanger. More specifically, the first air pre-heater is accommodated in the device downstream viewed from the steam turbine. This has the advantage that the steam in the steam line is first passed through the steam turbine at a high temperature, which improves the transfer of power to the steam turbine axle.

Furthermore, the device comprises a second air pre-heater, which brings the gases coming from the gas turbine into heat-exchanging contact with the gases coming from the gas compressor. In this case as well, it is possible to achieve an improved efficiency of the device via heat transfer from the gases coming from the gas turbine (having a high temperature) to the gases coming from the gas compressor (having a lower temperature).

In this last embodiment, the second air pre-heater is accommodated in the device upstream viewed from the at least one heat exchanger and, more specifically, the second air pre-heater is accommodated in the device downstream viewed from the first air pre-heater.

Furthermore, the steam converter may comprise a steam superheater, which, in operation, is in heat-exchanging contact with the gases coming from the gas turbine. In this case, it is also possible to achieve a further heating of the steam in the steam line before passing this steam through the steam turbine. Also as a result thereof, the efficiency of the device is improved.

More specifically, the gas turbine is in flow connection with the combustion chamber in order to pass the gases coming from the gas turbine to the combustion chamber.

More particularly, the steam turbine and gas compressor are positioned on the same axle.

Furthermore, the gas turbine unit (A) comprises a blow-off valve which is arranged upstream of the at least one heat exchanger for blowing off the gases coming

from the gas compressor in a controlled manner.

In this case, the blow-off valve may be accommodated in the gas turbine unit (A) downstream viewed from the first air pre-heater or in the gas turbine unit (A) upstream viewed from the second air pre-heater or in the gas turbine unit (A) upstream viewed from the gas turbine.

Furthermore, the compressor may consist of several compressors situated parallel to each other and/or in series and the turbine may consist of several turbines situated parallel to each other and/or in series and the steam turbine may consist of several steam turbines situated parallel to each other and/or in series.

The efficiency of the steam turbine unit may furthermore be improved by the fact that, with a specific embodiment according to the invention, in operation, the water flowing through the steam line is in heat-exchanging contact with the air flowing through the intermediate cooler downstream of the condenser. More particularly, the heat-exchanging contact takes place according to the co-current principle or according to the countercurrent principle.

With other embodiments of the steam and gas turbine device according to the invention, the steam turbines may be impulse steam turbines or radial steam turbines, the gas compressors may be centrifugal gas compressors or axial gas compressors and the steam generator may be a once-through waste heat boiler.

The invention will now be explained with reference to a drawing, in which:

Figs. 1a-1b-2-3-4-5-6-7-8-9 successively show various embodiments of a steam and gas turbine device according to the invention.

In order to aid the understanding of the invention, similar components illustrated in the various figures will be denoted by the same reference numerals in the following description of the figures.

In the embodiments of a steam and gas turbine device according to the invention illustrated in Fig. 1a and 1b, the gas compressor 1 is shown as being positioned on one axle 4, 10, together with the steam turbine 9. By means of this configuration of a steam and gas turbine device according to the invention, the power generated by the steam turbine unit (B) is transferred to the gas turbine 3 via the axle 4, 10 and the gas compressor 1, so that all of the power generated by this device is transferred to one axle 5 in order to drive the load 12.

With all the embodiments of the steam and gas turbine device according to the invention described here, the steam turbines may be impulse steam turbines or radial steam turbines, the gas compressors may be centrifugal gas compressors or axial gas

compressors and the steam generator may be a once-through waste heat boiler.

The characteristic feature of the steam and gas turbine device is the mechanical disengagement of the gas compressor 1 and the gas turbine 3 which are not positioned on the same axle, but are positioned on different axles 4 and 5, respectively.

5 This mechanical disengagement of the gas compressor 1 and the gas turbine 3 renders the use of additional, complicated coupling mechanisms obsolete.

In this embodiment, reference numeral 2 denotes an arbitrary combustion chamber/oven which is fed with fuel which is supplied via the inlet 2a. Actually, the combustion chamber/oven 2 may be used for any factory process which has a hot (waste) stream as output. The combustion chamber 2 may be of any desired design and may be, for example, a combustion engine or an incinerator, depending on the technical installation which is used in the steam and gas device according to the invention.

10 Reference numeral 20 denotes a heat exchanger which brings the hot (waste) stream coming from the combustion chamber 2, such as combustible flue gases, into heat-exchanging contact with the gases coming from the gas compressor in compressed form via the outlet 1b.

Following the heat-exchanging contact between the combustible flue gases coming from the combustion chamber 2 and the compressed gases coming from the gas compressor 1, the latter gases are passed into the gas turbine 3 via the inlet 3a. The compressed gases which have been heated by the heat-exchanging contact in the heat exchanger 10 drive the gas turbine 3, as a result of which the latter will start to rotate, together with the gas turbine axle 5, and will thus drive a load, which is denoted by reference numeral 12 in the figure.

20 The heated gases leave the gas turbine 3 via the outlet 3b and are passed to the steam generator 8. Similarly, the combustible flue gases are passed from the heat exchanger 20 through the steam generator 8 via the outlet 2b.

In the steam generator 8, the hot gas turbine gases and the hot combustible flue gases are brought into heat-exchanging contact with the water flowing through the closed steam line 6, so that steam is generated, by means of which the steam turbine 9 is driven.

30 In this steam and gas turbine device, the gas compressor 1 is positioned on one axle 4, 10 together with the steam turbine 9. Using this configuration, the power generated by the steam turbine unit (B) is transferred to the gas turbine 3 via the axle 4, 10 and the gas compressor 1, so that all the power generated by this device is transferred to one axle 5 in order to drive the load 12.

By using the heat exchanger 20 in this embodiment of the steam and gas turbine device according to the invention, the entire unit can easily be coupled to installations which already contain a combustion unit. As a result thereof, the steam and gas turbine device requires few modifications as the heat exchanger 20 can easily be
5 connected to the hot (waste) stream outlet of the combustion chamber 2 which is already present.

In the embodiment as illustrated in Figs. 1a and 1b, the steam converter comprises a steam generator 8 which, in operation, brings the steam/water line 6 into heat-exchanging contact with the combustible flue gases coming from the heat exchanger
10 20 via the outlet 2b and also with the gases coming from the gas turbine 3 via the outlet 3b.

One other more efficient embodiment of the steam and gas turbine device is illustrated in Fig. 2, wherein the reference numeral 30 comprises a first air pre-heater which brings the steam line 6 through which the compressed gases coming from the gas
15 compressor 1 pass into heat-exchanging contact with the outlet 1b. Here, the waste heat of the steam passed through the steam/water line 6 is emitted to the gases compressed by the gas compressor 1 and flowing through the outlet 1b, so that these gases are passed to the heat exchanger 20 in preheated and compressed form via the outlet 1b.

As is illustrated in Fig. 2, the first air pre-heater 30 is accommodated in the
20 device downstream viewed from the steam generator 8 and upstream viewed from the heat exchanger 20. More specifically, as is illustrated in Fig. 2, the first air pre-heater 30 brings the outlet 1b of the gas compressor into heat-exchanging contact with the steam/water line 6 downstream viewed from the steam turbine 9.

In Fig. 3, a particular form of the embodiment of the steam and gas turbine
25 device according to the invention shown in Fig. 2 is illustrated. In this Fig. 3, reference numeral 40 denotes a second air pre-heater which brings the gases coming from the gas turbine 3 into heat-exchanging contact with the gases coming from the gas compressor 1 and more particularly the gases coming from the first air pre-heater 30. In this embodiment, the gases coming from the gas turbine 3 are passed to the steam
30 converter/steam generator 8 (see outlet 3b) after they have passed through the second air pre-heater 40. An improved energy efficiency is achieved in this case as well in the form of heat recovery, due to the fact that the compressed gases which are released from the gas compressor 1 via the outlet 1b are preheated due to the transfer of energy from the gases from the gas turbine 3.

35 The embodiment in Fig. 4 is derived from the embodiment from Fig. 1,

albeit that the steam converter also comprises a steam superheater 50 in addition to a steam generator 8, so that, in operation, the steam/water line 6 is in heat-exchanging contact with the gases coming from the gas turbine 3. In this embodiment, these gases are blown off via the outlet 3b.

5 In this way, a further heating of the steam passed through the steam/water line 6 and generated in the steam generator 8 is further heated in a supercritical manner in order to achieve a more efficient energy conversion in the steam turbine 9. Optionally, an additional water spray injector may be provided with this steam superheater 50 in order to introduce a water mist into the steam/water line 6, so that extra steam is
10 generated which can be passed to the steam turbine 9.

A combination of the embodiments from Figs. 2 and 4 is illustrated in Fig. 5. In this combined embodiment, both a first air pre-heater 30 and a steam superheater 50 are implemented in which the gases coming from the gas turbine 3 are blown off via the outlet 3b. In this embodiment, the steam conversion in the steam generator 8 only
15 takes place by heat transfer between the combustible flue gases (outlet 2b) coming from the combustion chamber 2 and the heat exchanger 20 and the water passed to the steam generator 8 via the pump 7.

In Fig. 6, which refers back to the embodiment according to Fig. 1 in which the gases coming from the gas turbine 3 are not first passed to the steam converter 8, but
20 are first passed to the combustion chamber 2, where they are fed to the combustion process. Thus, the gases fed to the combustion chamber 2 via the gas turbine 3 are also passed through the steam generator 8 as combustible flue gases via the outlet 2b. In this embodiment, the steam generator is thus only supplied with combustible flue gases via the outlet 2b which are composed of gases which are passed to the combustion chamber
25 2 via the inlet 2a or via the outlet 3b of the gas turbine 3.

A combination of the embodiments from Figs. 2 and 6 is illustrated in Fig. 7. Also in this embodiment, the gases coming from the gas turbine 3 are passed to the combustion chamber 2 via the outlet 3b and the steam/water line 6 is in heat-exchanging contact with the outlet 1b downstream viewed from the steam turbine 9, through which the
30 compressed gases coming from the gas compressor 1 are passed.

A combination of the embodiments from Figs. 2, 5 and 6 is illustrated in Fig. 8. In this embodiment, the steam superheater 50 is implemented which is in heat-exchanging contact with the gases coming from the gas turbine 3 which are passed into the combustion chamber 2 via the outlet 3b and of which only the combustible flue gases
35 then come into heat-exchanging contact with the steam/water line 6 in the steam

generator 8. After heating in the steam superheater 50 and after the steam turbine 9, the steam passed from the steam generator 8 to the steam superheater 50 is, in the first air pre-heater 30, again brought into heat-exchanging contact with the compressed gases which leave the gas compressor 1 via the outlet 1b.

5 Thus, a last waste heat which is still present in the steam is efficiently transferred to the compressed gases before being passed through the heat exchanger 20.

 It will be clear that, with the illustrated embodiments of the steam and gas turbine device according to the invention, by the implementation of the heat exchanger 20, the entire device can easily be used with any installation where a hot (waste) stream
10 is generated in an external combustion process or any other process.

 It should furthermore be noted that the heat-exchanging contact in the first and second air pre-heaters 30 and 40, respectively, between the compressed gases from the gas compressor 1 and the steam in the steam/water line 6, or in the steam generator 8 and the steam superheater 50, respectively, between the steam in the steam/water line
15 6 and the gases coming from the gas turbine 3 / the combustible flue gases coming from the heat exchanger 20 can take place according to the co-current principle or according to the countercurrent principle.

 In the figures, reference numeral 21 denotes a blow-off valve which, is arranged in any case in all embodiments upstream of the gas turbine 3 (see all figures
20 and in particular Fig. 1a) and preferably upstream of the at least one heat exchanger 20 (see all figures and in particular Figs. 1b and 9) and serves for blowing off gases coming from the gas compressor 1 via the outlet 1c. Preferably, the blow-off valve 21 is accommodated in the gas turbine unit (A) downstream viewed from the first air pre-heater 30 (Figs. 2, 3, 5, 7, 8), whereas the blow-off valve 21 is accommodated in the gas turbine
25 unit (A) upstream viewed from the second air pre-heater 40, as is illustrated in Fig. 3. Optionally, the device may comprise a blow-off valve 21 which is arranged upstream of the gas turbine 3 in order to blow off the gases coming from the gas compressor 1.

 In addition to the various variants of a steam and gas turbine device according to the invention illustrated in the Figs. 1a-1b-2-3-4-5-6-7-8-9 the illustrated
30 variants may be supplemented with an additional advantageous aspect, in which the blow-off valve 21 and the outlet 1c are in flow connection with the combustion chamber 2. With this embodiment, which can be applied to each Fig. 1 to 9, the blow-off valve 21 is returned to the combustion chamber 2 via the outlet 1c. Thus, the gases coming from the gas compressor 1 and which are usually blown off via the blow-off valve 21 and the outlet
35 1c, are now returned to the combustion chamber 2. These additional embodiments

provide an additional more efficient combustion and thus energy conversion in the combustion chamber 2 and ultimately also in the heat exchanger 20.

In addition, Fig. 9 shows an embodiment in which the combustion chamber 2 and the heat exchanger 20 have been integrated with each other such that they form a
5 single entity, resulting in a more efficient energy conversion. The integration of the combustion chamber 2 and the heat exchanger 20 with each other to form a single entity may also be used in the other embodiments which are illustrated in Figs. 1a-1b-2-3-4-5-6-7-8.

CLAIMS

1. Steam and gas turbine device comprising a gas turbine unit (A) and a steam turbine unit (B), wherein the gas turbine unit (A) is partly composed of at least the following:
- 5
- a gas compressor (1) positioned on a compressor axle (4), a combustion chamber (2) and a gas turbine (3) positioned on a gas turbine axle (5); and wherein the steam turbine unit (B) is partly composed of the following:
- 10
- a closed steam line (6) in which at least the following are accommodated:
 - a pump (7); a steam converter (8; 50), which, in operation, is in heat-exchanging contact with the gases (3b) coming from the gas turbine (3); a steam turbine (9) positioned on a steam turbine axle (10); as well as a condenser (11) and wherein, in operation, the steam turbine (9) drives the gas compressor (1) of the gas turbine unit (A), wherein the gas turbine unit (A) furthermore comprises at least one heat exchanger (20)
- 15
- which brings the combustible flue gases coming from the combustion chamber (2) into heat-exchanging contact with the gases coming from the gas compressor (1) and wherein the gas turbine unit (A) comprises a blow-off valve (21) which is arranged upstream of the gas turbine (3) for blowing off the gases coming from the gas compressor (1).
2. Steam and gas turbine device according to Claim 1, characterized in that
- 20
- the blow-off valve (21) is arranged upstream of the at least one heat exchanger (20).
3. Steam and gas turbine device according to Claim 1 or 2, characterized in that the steam converter (8; 50) comprises a steam generator (8), which, in operation, is in heat-exchanging contact with the combustible flue gases coming from the combustion chamber (2).
- 25
4. Steam and gas turbine device according to one or more of the preceding claims, characterized in that the device furthermore comprises a first air pre-heater (30), which brings the steam line (6) into heat-exchanging contact with the gases coming from the gas compressor (1).
5. Steam and gas turbine device according to Claim 4, characterized in that
- 30
- the first air pre-heater (30) is accommodated in the device downstream viewed from the steam generator (8) and upstream viewed from the at least one heat exchanger (20).
6. Steam and gas turbine device according to Claim 2 in combination with Claim 5, characterized in that the blow-off valve (21) is accommodated in the gas turbine unit (A) downstream viewed from the first air pre-heater (30).
- 35
7. Steam and gas turbine device according to Claim 4 or 5, characterized in

that the first air pre-heater (30) is accommodated in the device downstream viewed from the steam turbine (9).

8. Steam and gas turbine device according to one or more of the preceding claims, characterized in that the device furthermore comprises a second air pre-heater (40) which brings the gases coming from the gas turbine (3) into heat-exchanging contact with the gases coming from the gas compressor (1).

9. Steam and gas turbine device according to Claim 6 in combination with Claim 8, characterized in that the blow-off valve (21) is accommodated in the gas turbine unit (A) upstream viewed from the second air pre-heater (40).

10. Steam and gas turbine device according to Claim 8, characterized in that the second air pre-heater (40) is accommodated in the device upstream viewed from the at least one heat exchanger (20).

11. Steam and gas turbine device according to Claim 10, characterized in that the second air pre-heater (40) is accommodated in the device downstream viewed from the first air pre-heater (30).

12. Steam and gas turbine device according to one or more of the preceding claims, characterized in that the steam converter (8; 50) comprises a steam superheater (50), which, in operation, is in heat-exchanging contact with the gases (3b) coming from the gas turbine (3).

13. Steam and gas turbine device according to one or more of the preceding claims, characterized in that the gas turbine (3) is in flow connection with the combustion chamber (2) in order to pass the gases (3b) coming from the gas turbine to the combustion chamber (2).

14. Steam and gas turbine device according to one or more of the preceding claims, characterized in that the gas compressor (1) and the steam turbine (9) are positioned on the same axle (4, 10).

15. Steam and gas turbine device according to one or more of the preceding claims, characterized in that the blow-off valve (21) is connected to the combustion chamber (2).

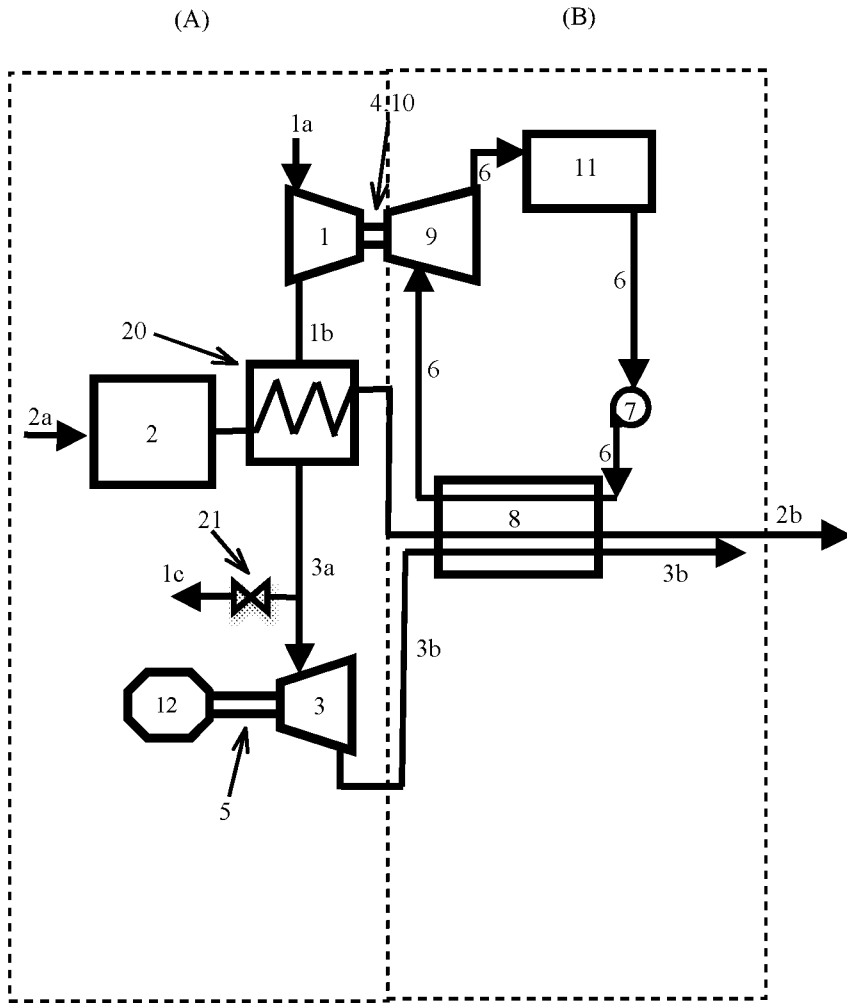


Fig. 1a

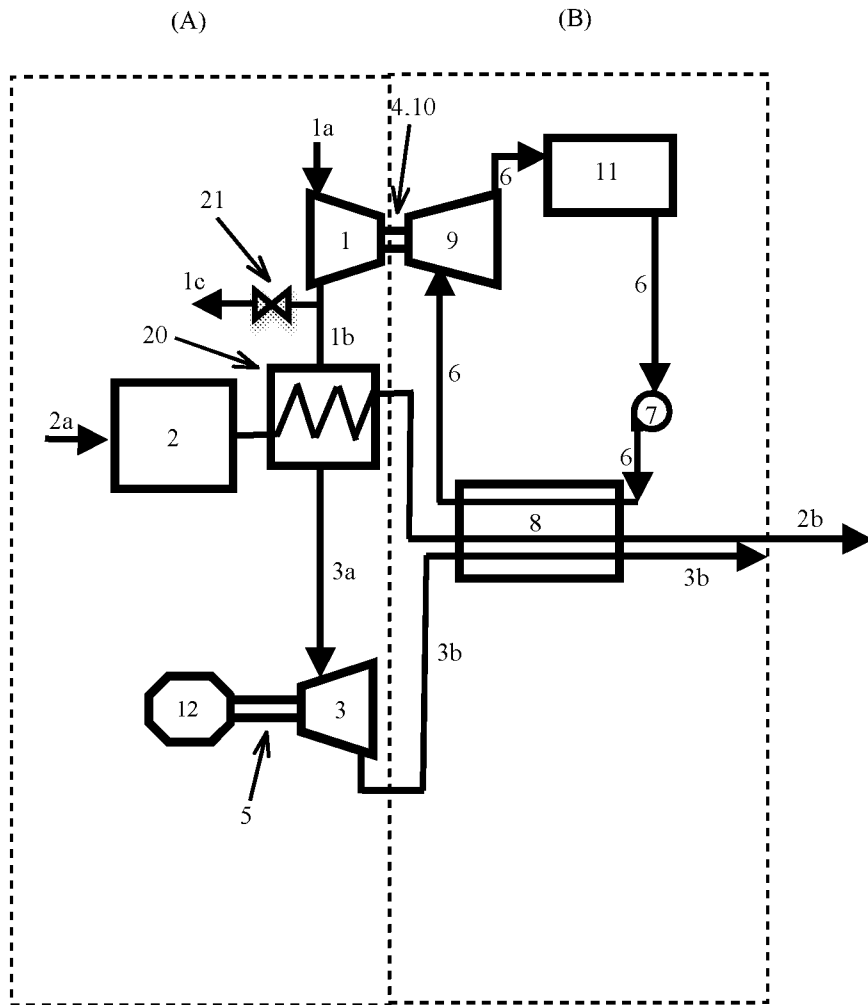


Fig. 1b

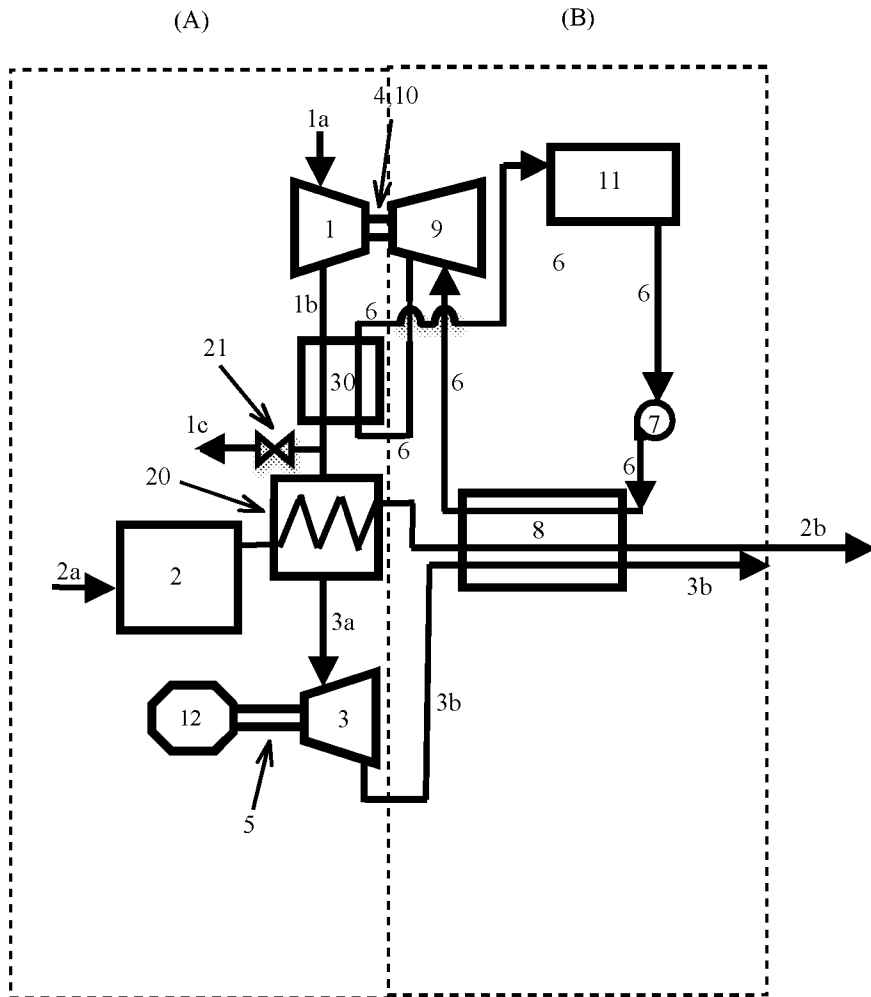


Fig. 2

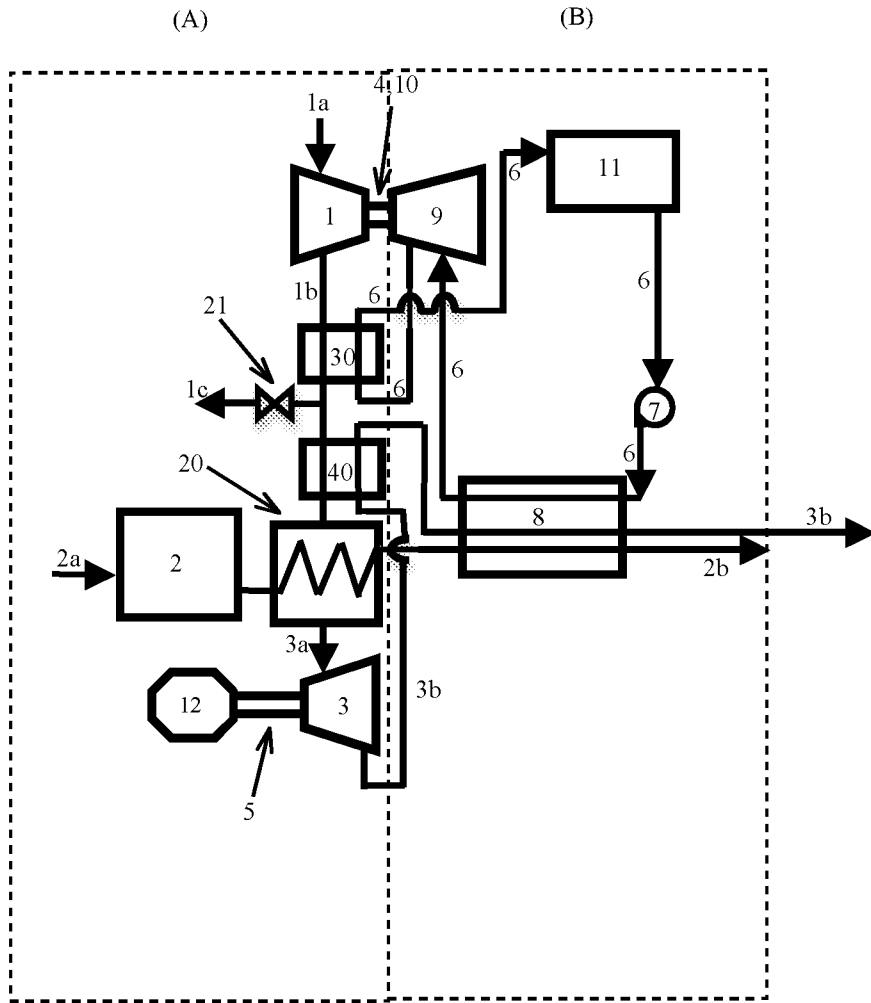


Fig. 3

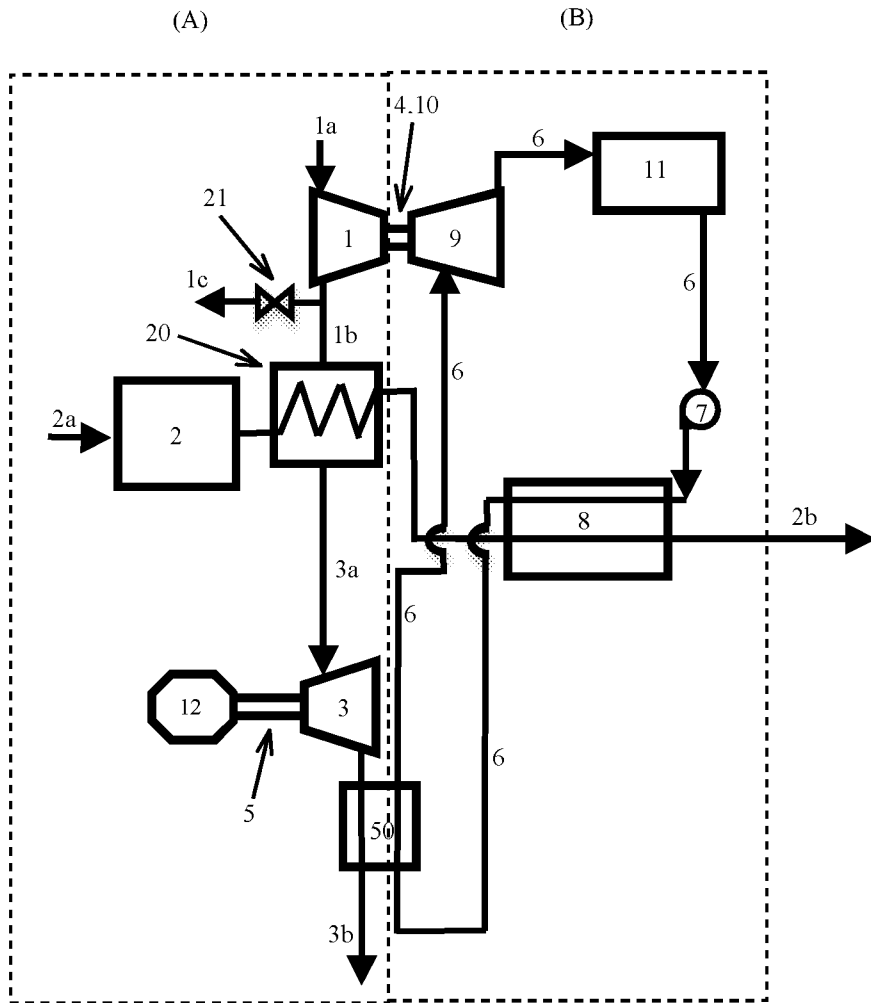


Fig. 4

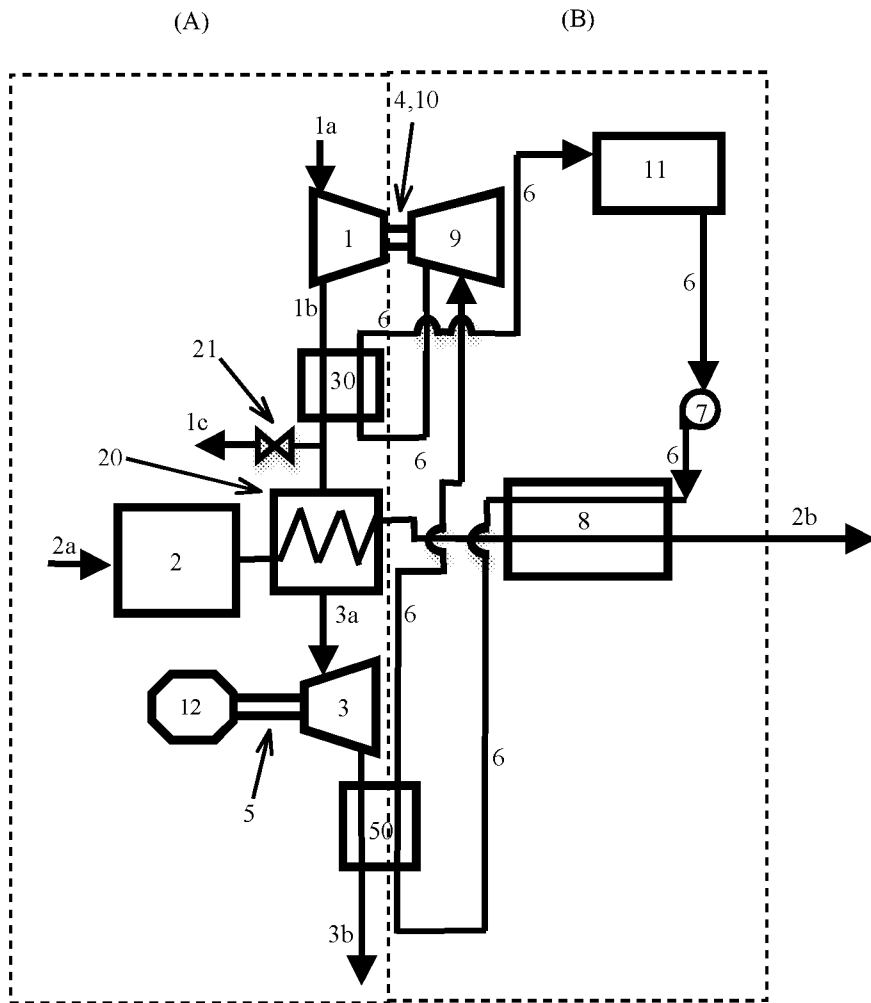


Fig. 5

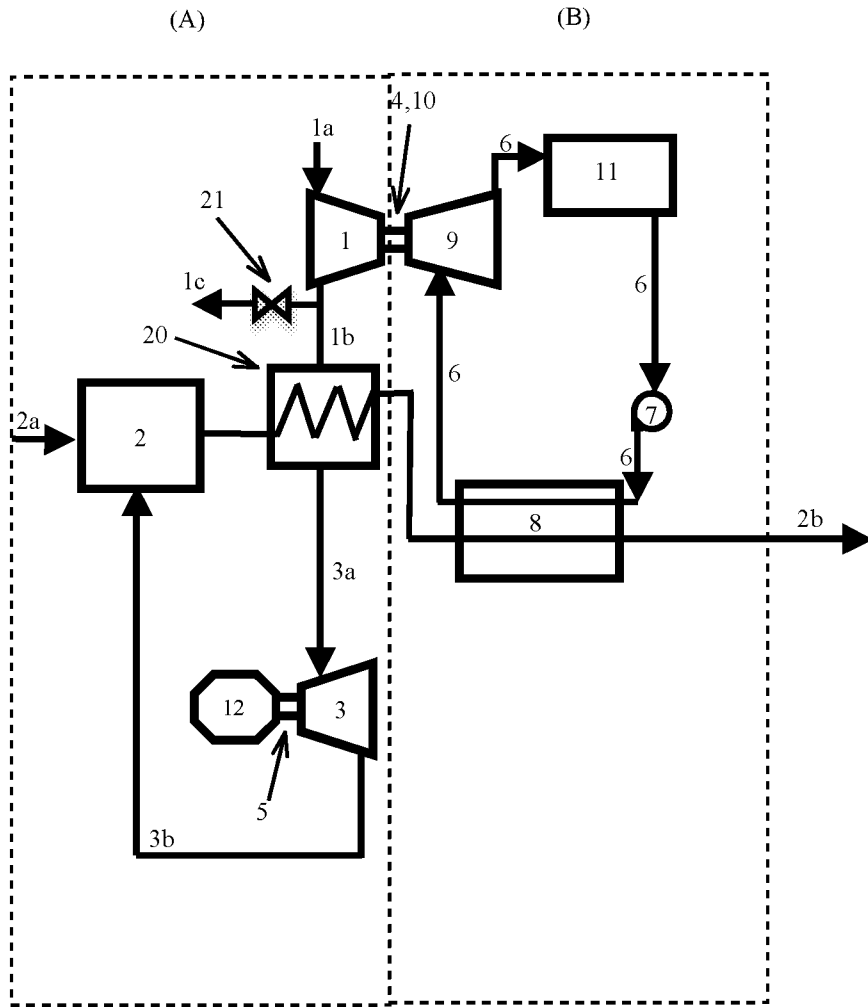


Fig. 6

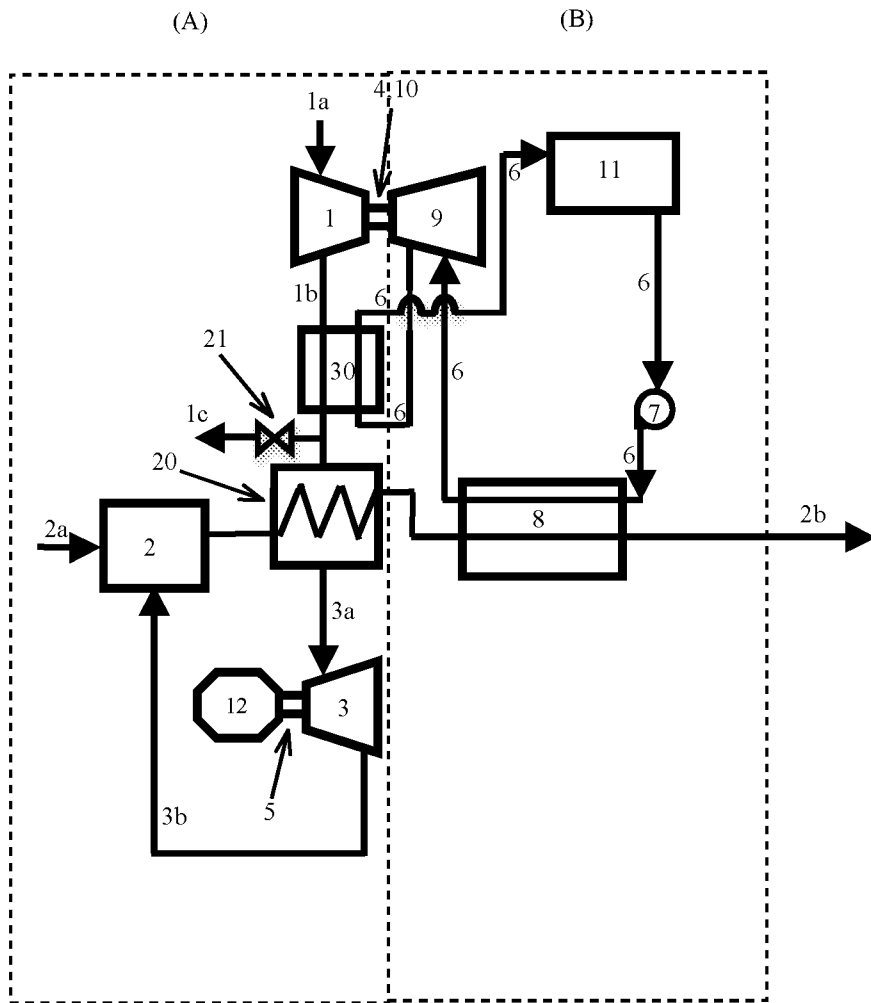


Fig. 7

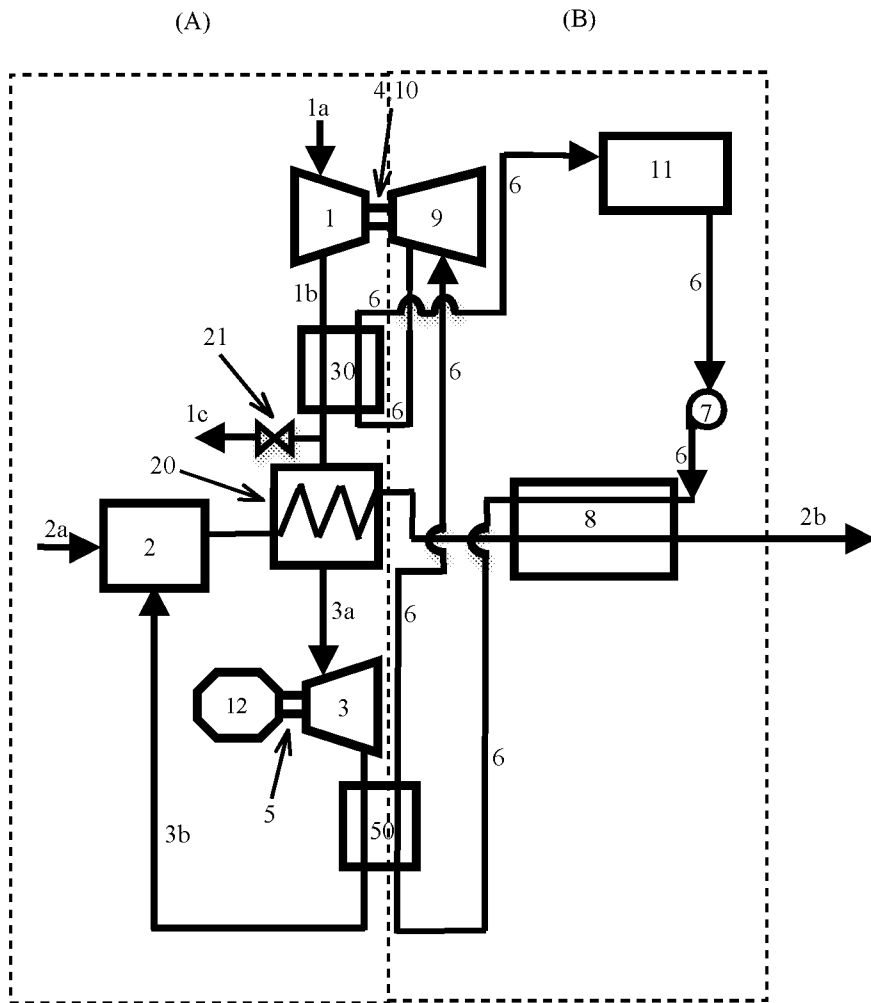


Fig. 8

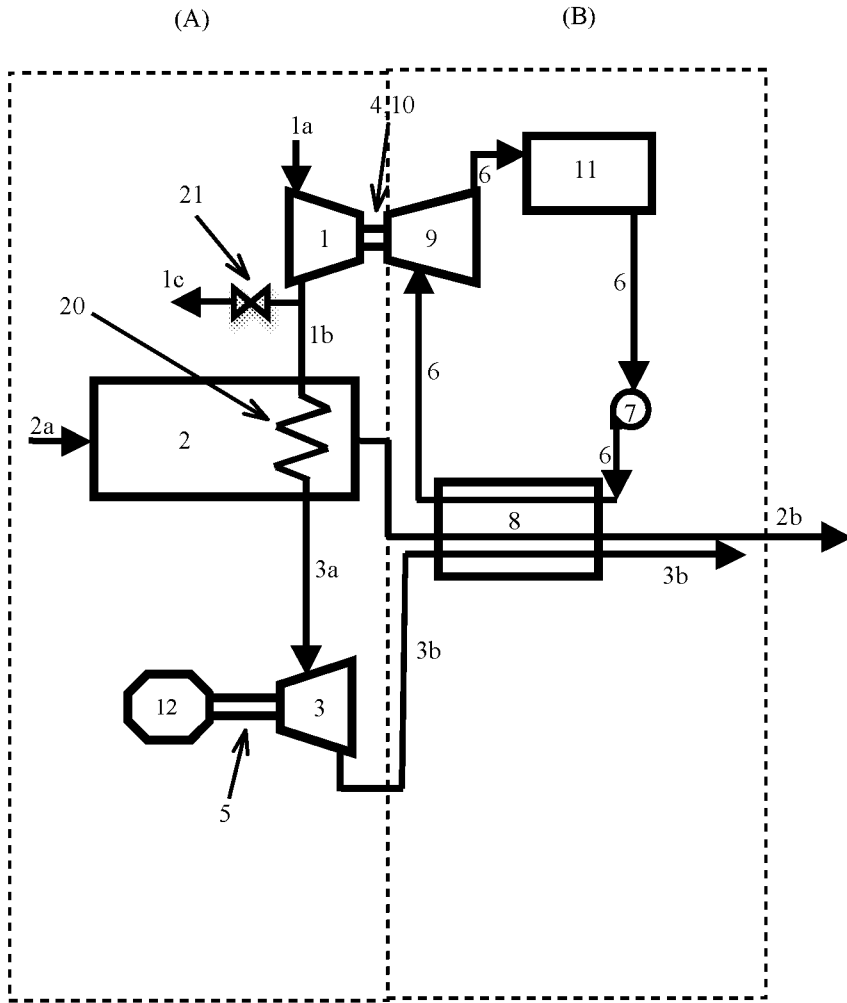


Fig. 9

INTERNATIONAL SEARCH REPORT

International application No
PCT/NL2017/050505

A. CLASSIFICATION OF SUBJECT MATTER
INV. F01K23/10 F01K23/16
ADD.
According to International Patent Classification (IPC) or to both national classification and IPC

B. FIELDS SEARCHED
Minimum documentation searched (classification system followed by classification symbols)
F01K

Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched

Electronic data base consulted during the international search (name of data base and, where practicable, search terms used)
EPO-Internal, WPI Data

C. DOCUMENTS CONSIDERED TO BE RELEVANT

| Category* | Citation of document, with indication, where appropriate, of the relevant passages | Relevant to claim No. |
|-----------|---|-----------------------|
| A | DE 10 2010 033659 A1 (DAIMLER AG [DE]) 9 February 2012 (2012-02-09) paragraphs [0034] - [0054], [0060] - [0063]; figure 4 | 1-15 |
| Y | GB 671 702 A (OERLIKON MASCHF) 7 May 1952 (1952-05-07) page 2, line 5 - line 95; figure 2 | 1-3, 12-15 |
| Y | JP H08 74517 A (TOSHIBA CORP) 19 March 1996 (1996-03-19) paragraph [0010]; figures 1-5 | 1-3, 12-15 |
| A | US 2005/204723 A1 (OUWERKERK HENK [NL]) 22 September 2005 (2005-09-22) the whole document | 1-15 |

Further documents are listed in the continuation of Box C.

See patent family annex.

* Special categories of cited documents :

- "A" document defining the general state of the art which is not considered to be of particular relevance
- "E" earlier application or patent but published on or after the international filing date
- "L" document which may throw doubts on priority claim(s) or which is cited to establish the publication date of another citation or other special reason (as specified)
- "O" document referring to an oral disclosure, use, exhibition or other means
- "P" document published prior to the international filing date but later than the priority date claimed

"T" later document published after the international filing date or priority date and not in conflict with the application but cited to understand the principle or theory underlying the invention

"X" document of particular relevance; the claimed invention cannot be considered novel or cannot be considered to involve an inventive step when the document is taken alone

"Y" document of particular relevance; the claimed invention cannot be considered to involve an inventive step when the document is combined with one or more other such documents, such combination being obvious to a person skilled in the art

"&" document member of the same patent family

| | |
|---|--|
| Date of the actual completion of the international search 20 November 2017 | Date of mailing of the international search report 30/11/2017 |
|---|--|

| | |
|--|---|
| Name and mailing address of the ISA/ European Patent Office, P.B. 5818 Patentlaan 2 NL - 2280 HV Rijswijk Tel. (+31-70) 340-2040, Fax: (+31-70) 340-3016 | Authorized officer Röberg, Andreas |
|--|---|

INTERNATIONAL SEARCH REPORT

Information on patent family members

| |
|---|
| International application No PCT/NL2017/050505 |
|---|

| Patent document cited in search report | Publication date | Patent family member(s) | Publication date |
|--|------------------|-------------------------|------------------|
| DE 102010033659 A1 | 09-02-2012 | NONE | |
| ----- | | | |
| GB 671702 | A | 07-05-1952 | |
| | | BE 490890 A | 20-11-2017 |
| | | DE 821733 C | 19-11-1951 |
| | | FR 1017180 A | 03-12-1952 |
| | | GB 671702 A | 07-05-1952 |
| ----- | | | |
| JP H0874517 | A | 19-03-1996 | NONE |
| ----- | | | |
| US 2005204723 | A1 | 22-09-2005 | |
| | | AT 421632 T | 15-02-2009 |
| | | AU 2003235441 A1 | 27-10-2003 |
| | | CA 2482102 A1 | 23-10-2003 |
| | | CN 1656304 A | 17-08-2005 |
| | | EP 1492941 A1 | 05-01-2005 |
| | | JP 2005522621 A | 28-07-2005 |
| | | KR 20050003361 A | 10-01-2005 |
| | | NL 1020350 C2 | 13-10-2003 |
| | | US 2005204723 A1 | 22-09-2005 |
| | | WO 03087543 A1 | 23-10-2003 |
| ----- | | | |